

Flame Speed Problem

Thursday, October 12, 2017 11:53 AM

Estimate the laminar flame speed of a stoichiometric propane-air mixture using the simplified theory result. Use the global one-step reaction mechanism below to estimate the mean reaction rate.

$$d[\text{C}_3\text{H}_8]/dt = -k_G [\text{C}_3\text{H}_8]^{0.1} [\text{O}_2]^{1.6}$$

$$k_G = 4.836 \cdot 10^9 \cdot \exp(-15,098/T) \quad (\text{kmol/m}^3)^{-0.75} / \text{s}$$

$$T_{\text{ad},\text{C}_3\text{H}_8} = 2260 \text{ K}$$

$$\text{MW}_{\text{O}_2} = 32.0 \text{ kg/kmol}$$

$$\text{MW}_{\text{N}_2} = 28.01 \text{ kg/kmol}$$

$$\text{MW}_{\text{C}_3\text{H}_8} = 44.10 \text{ kg/kmol}$$

$$\bar{T} = \frac{1}{2} \left(\frac{T_b + T_u}{2} + T_b \right)$$

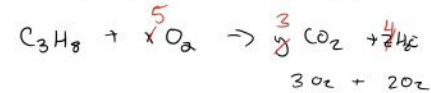
$$T_u = 300 \text{ K} \quad T_b = 2260 \text{ K}$$

$$\bar{T} = \frac{1}{4} (2560) + \frac{1}{2} (2260) = 1770 \text{ K}$$

$$Y_{F,\delta} = 0$$

$$Y_{\text{ox},\delta} = 0$$

$$\bar{Y}_{\text{C}_3\text{H}_8} = \frac{1}{2} (Y_{\text{C}_3\text{H}_8,u} + 0) = \frac{1}{2} Y_{\text{C}_3\text{H}_8,u}$$



$$Y_{\text{C}_3\text{H}_8,u} = \frac{1 (\text{MW}_{\text{C}_3\text{H}_8})}{1 (\text{MW}_{\text{C}_3\text{H}_8}) + 5 (\text{MW}_{\text{O}_2}) + \frac{79}{21} 5 (\text{MW}_{\text{N}_2})}$$

$$= 0.06015$$

$$\bar{Y}_{\text{C}_3\text{H}_8} = \frac{1}{2} Y_{\text{C}_3\text{H}_8,u} = 0.0301$$

$$\bar{Y}_{\text{O}_2} = \frac{1}{2} (Y_{\text{O}_2,u} + Y_{\text{O}_2,B}) = \frac{1}{2} Y_{\text{O}_2,u}$$

$$Y_{\text{O}_2,u} = \frac{5 (32)}{\sum N_i \text{MW}_i}$$

$$= 0.219$$

$$\bar{Y}_{\text{O}_2} = 0.1095$$

$$a = \frac{k_G \text{ oxidizer}}{k_G \text{ fuel}} = \frac{5 \text{ MW}_{\text{O}_2} + \frac{79}{21} 5 \text{ MW}_{\text{N}_2}}{\text{MW}_F} = 15.625$$

$$\dot{w}_c = \frac{d[\text{C}_3\text{H}_8]}{dt} = -k_G [\text{C}_3\text{H}_8]^{0.1} [\text{O}_2]^{1.65}$$

$$b. = 4.836 \times 10^9 \left(\frac{-15098}{T} \right)$$

$$= 9,55 \times 10^5 \left(\frac{\text{kmol}}{\text{m}^3} \right)^{-0.75} \frac{1}{5}$$

$$[C_3H_8] = \left(\frac{P}{R_0 T} \right) \frac{Y_{C_3H_8}}{MW_{C_3H_8}} (\bar{m}w)$$

$$\bar{p} = \bar{m}w \left(\frac{P}{R_0 T} \right)$$

$$\bar{m}w = \frac{1}{2} (MW_B + MW_U)$$

$$MW_U = \frac{5(MW_{O_2}) + 1(MW_{C_3H_8}) + 5 \frac{79}{21} MW_{N_2}}{1 + 5 + 5 \frac{79}{21}}$$

$$= 29.46 \text{ kg/kmol}$$

$$MW_B = \frac{3(MW_{CO_2}) + 4(MW_{H_2O}) + 5 \frac{79}{21} MW_{N_2}}{3 + 4 + 5 \frac{79}{21}}$$

$$= 28.32 \text{ kg/kmol}$$

$$\bar{m}w = \frac{1}{2} (MW_U + MW_B) = 28.89 \text{ kg/kmol}$$

$$\bar{p} = \bar{m}w \left(\frac{P}{R_0 T} \right) = 28.89 \left(\frac{101325 \text{ Pa}}{(8315) 1770} \right)$$

$$= 0.1989 \text{ kg/m}^3$$

$$[C_3H_8] = \frac{Y_{C_3H_8}}{MW_{C_3H_8}} \bar{p} = 0.0001360 \text{ kmol/m}^3$$

$$[O_2] = \frac{Y_{O_2}}{MW_{O_2}} \bar{p} = 0.0006806 \text{ kmol/m}^3$$

$$\bar{w}_{C_3H_8} = -k_c [C_3H_8]^{0.1} [O_2]^{1.65}$$

$$= -2.331 \text{ kmol/m}^3\text{-s}$$

$$\dot{m}_F'' = \bar{w}_{C_3H_8} MW_{C_3H_8} = -102.06 \frac{\text{kg}}{\text{m}^2\text{-s}}$$

$$\sim 1280 \text{ K} \quad k = 0.0809 \text{ kJ/m-s-K}$$

$$c_p = 1.186 \text{ kJ/kg-K}$$

$$\rho_U = 1.16 \text{ kg/m}^3$$

$$\bar{T} = \frac{1}{2} (T_B + T_U) = 1280 \text{ K}$$

$$\alpha = \frac{k}{\rho_U c_p} = 5.89 \times 10^{-5} \text{ m}^2/\text{s}$$

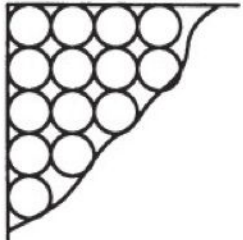
$$Sc = \left[-2\alpha (a+1) \frac{\dot{m}_{C_3H_8}''}{\rho_U} \right]^{1/2}$$

$$= 0.416 \text{ m/s} \text{ or } \boxed{41.6 \text{ cm/s}}$$

Quenching Problem

Tuesday, October 24, 2017 10:48 AM

Consider the design of a laminar-flow, adiabatic, flat-flame burner consisting of a square arrangement of thin walled tubes as illustrated in the sketch below. Fuel-air mixture flows through both the tubes and the interstices between the tubes. It is desired to operate the burner with a stoichiometric methane-air mixture exiting the burner tubes at 300 K and 5 atm.



Burner tube layout

$$R_u = 0.08206 \text{ atm}\cdot\text{m}^3/\text{kmol}\cdot\text{K}$$

$$MW_{\text{methane}} = 16.04 \text{ kg/kmol}$$

$$MW_{\text{air}} = 28.85 \text{ kg/kmol}$$

A) Determine the mixture mass flowrate per unit cross-

sectional area at the design condition.

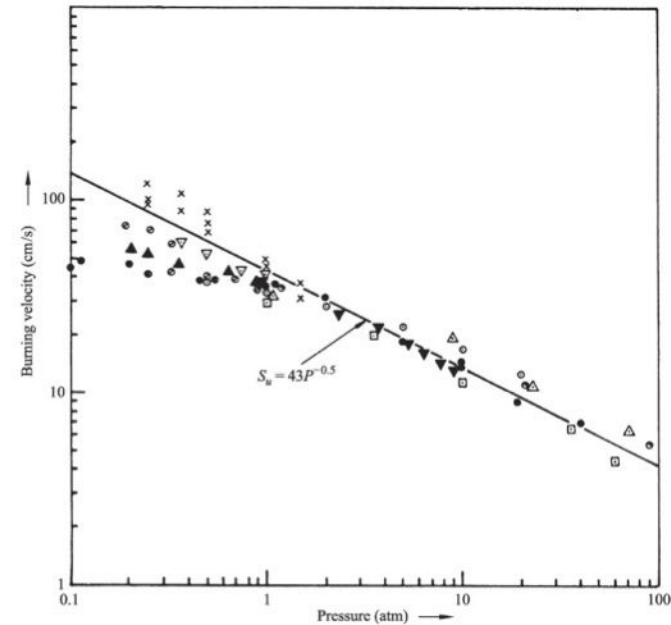


Figure 8.14 Effect of pressure on laminar flame speeds of stoichiometric methane-air mixtures for $T_u = 16\text{--}27^\circ\text{C}$.
SOURCE: Reprinted with permission, Elsevier Science, Inc., from Ref. [19], © 1972, The Combustion Institute.

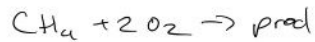
$$\dot{m}'' = \rho_0 S_L$$

$$S_L (300 \text{ K}, 5 \text{ atm}) = 43 / \sqrt{5}$$

$$= 19.2 \text{ cm/s}$$

$$\rho_u = \frac{P}{(R_u/MW_{mix})T_u}$$

$$MW_{mix} = X_F MW_F + (1-X_C) MW_{O_2}$$



$$X_F = \frac{1}{(1+2 + 2 \frac{32}{16})} = \frac{1}{10.52} = 0.095$$

$$MW_{mix} = 27.6 \text{ kg/kmol} \Rightarrow \rho_u = 5.61 \frac{\text{kg}}{\text{m}^3}$$

$$\dot{m}'' = 1.08 \frac{\text{kg}}{\text{s} \cdot \text{m}^2}$$

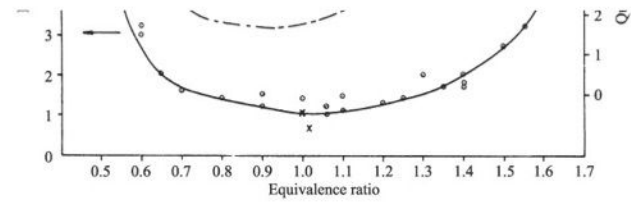


Figure 8.16 Flame thickness for laminar methane-air flames at atmospheric pressure. Also shown is the quenching distance.
SOURCE: Reprinted with permission, Elsevier Science, Inc., from Ref. [19], © 1972, The Combustion Institute.

$$\frac{d_1}{d_2} = \frac{2 \sqrt{5} \alpha_1 / S_{L1}}{2 \sqrt{5} \alpha_2 / S_{L2}}$$

State 1: $P = 5 \text{ atm}$
 $T_u = 300 \text{ K}$
 $S_L = 19.2 \text{ cm/s}$

State 2: $P = 1 \text{ atm}$
 $T_u = 300 \text{ K}$
 $S_L = 43 \text{ cm/s}$
 $d = 1.7 \text{ mm}$

$$d_1 = d_2 \frac{\alpha_1}{\alpha_2} \frac{S_{L2}}{S_{L1}}$$

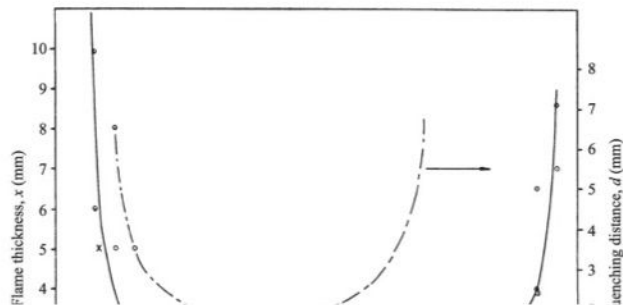
$$\alpha = T_u \bar{T}^{0.75} P^{-1}$$

$$\alpha = \frac{1}{P}$$

$$d_1 = d_2 \left(\frac{P_2}{P_1} \right) \frac{S_{L2}}{S_{L1}}$$

$$= 1.7 \text{ mm} \left(\frac{1 \text{ atm}}{5 \text{ atm}} \right) \left(\frac{43}{19.2} \right) = 0.76 \text{ mm}$$

1. Estimate the maximum tube diameter allowed so that flashback will be prevented.



$$Re = \frac{\rho U d S_L}{\mu} \quad \text{assume air prop.}$$
$$= 61.5$$

Flammability Limit Problem

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A full propane (C_3H_8) cylinder from a camp stove leaks its contents of 1.02 lb (0.464 kg) into a 12' x 14' x 8' (3.66 m x 4.27 m x 2.44 m) room at 20 C and 1 atm. After a long time, the fuel gas and the room air are well mixed. Will the room go boom in the event of a spark?

$$R_u = 0.08206 \text{ atm}\cdot\text{m}^3/\text{kmol}\cdot\text{K}$$

$$MW_{\text{propane}} = 44.094 \text{ kg}/\text{kmol}$$

$$MW_{\text{air}} = 28.85 \text{ kg}/\text{kmol}$$

$$X_F = \frac{V_F}{V}$$

$$P_F = \frac{n R T}{V} = \frac{m (R_u / MW_F) T}{V}$$

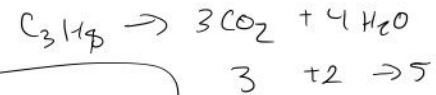
$$= \frac{(0.464) \left(\frac{0.08206}{44.094} \right) 293}{3.66 (4.27) (2.44)}$$

$$= 0.00663 \text{ atm}$$

$$X_F = \frac{P_F}{P} = 0.00663$$

$$X_{\text{air}} = 1 - X_F = 0.99337$$

$$\phi = \frac{(F/O_2)_{\text{act}}}{(F/O_2)_{\text{stoic}}} = \frac{0.00663 / 0.21 (0.99337)}{1/5}$$



$$\phi = 0.159$$

Table 8.4 Flammability limits, quenching distances, and minimum ignition energies for various fuels^a

Fuel	Flammability Limits		Stoichiometric Mass Air-Fuel Ratio	Quenching Distance		Minimum Ignition Energy	
	Φ_{min} (Lean or Lower Limit)	Φ_{max} (Rich or Upper Limit)		For $\Phi = 1$ (mm)	Absolute Minimum (mm)	For $\Phi = 1$ (10^{-2} J)	Absolute Minimum (10^{-2} J)
Acetylene, C_2H_2	0.19 ^b	∞	13.3	2.3	—	3	—
Carbon monoxide, CO	0.34	6.76	2.46	—	—	—	—
<i>n</i> -Decane, $C_{10}H_{22}$	0.36	3.92	15.0	2.1 ^c	—	—	—
Ethane, C_2H_6	0.50	2.72	16.0	2.3	1.8	42	24
Ethylene, C_2H_4	0.41	>6.1	14.8	1.3	—	9.6	—
Hydrogen, H_2	0.14 ^b	2.54 ^b	34.5	0.64	0.61	2.0	1.8
Methane, CH_4	0.46	1.64	17.2	2.5	2.0	33	29
Methanol, CH_3OH	0.48	4.08	6.46	1.8	1.5	21.5	14
<i>n</i> -Octane, C_8H_{18}	0.51	4.25	15.1	—	—	—	—
Propane, C_3H_8	0.51	2.83	15.6	2.0	1.8	30.5	26

^aSOURCE: Data from Ref. [21] unless otherwise noted.

^bZabetakis (U.S. Bureau of Mines, Bulletin 627, 1965).

^cChomiak [25].

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